

Shell boilers - Part 3: Design and calculation for pressure parts

Chaudières à tubes de fumée - Partie 3:
Conception et calcul des parties sous
pression

Großwasserraumkessel - Teil 3: Konstruktion
und Berechnung für drucktragende Teile

This corrigendum becomes effective on 3 July 2024 for incorporation in the official English version of the EN.



EUROPEAN COMMITTEE FOR STANDARDIZATION
COMITÉ EUROPÉEN DE NORMALISATION
EUROPÄISCHES KOMITEE FÜR NORMUNG

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1 Modification to 12.1, Thickness of straight tubes subject to external pressure

Replace the existing subclause 12.1 with the following:

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12.1 Thickness of straight tubes subject to external pressure

The wall thickness $e_t - c_1$ (ordered nominal thickness minus tolerance) of straight tubes ≤ 170 mm nominal outside diameter, subjected to external pressure, shall be greater or equal than the maximum given by Formulae (65), (66) or Table 7.

$$e_{ct,el.} = d_0 \sqrt[3]{\frac{pS_2(1-\nu^2)}{2E}} + c_{2,red.} \quad (65)$$

$$e_{ct,pl.} = \frac{pd_0}{1,6f} + c_2 \quad (66)$$

where

- $e_{ct,el}$ required wall thickness of straight tubes with reference to elastic buckling
- $e_{ct,pl}$ required wall thickness of straight tubes with reference to plastic deformation
- $c_2 = 0,75$ mm allowance for metal wastage
- $c_{2,red} = 0,3$ mm reduced allowance for metal wastage effective on elastic buckling
- ν Poisson's ratio
- S_2 safety factor ($S_2 = 3,0$; see 13.1.3).

NOTE For ferritic steel the Poisson's ratio $\nu = 0,3$ can be used.

Table 7 — Minimum thickness of tubes

Dimensions in millimetres

Nominal outside diameter	Minimum thickness $e_t - c_1$
$d_0 \leq 26,9$	1,90
$26,9 < d_0 \leq 54,0$	2,20
$54,0 < d_0 \leq 76,1$	2,50
$76,1 < d_0 \leq 88,9$	2,80
$88,9 < d_0 \leq 114,3$	3,15
$114,3 < d_0 \leq 139,7$	3,50
$139,7 < d_0 \leq 168,3$	3,99

”.