

Hydraulic machines, radial and axial - Performance conversion method from model to prototype

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NATIONAL FOREWORD

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English version

**Hydraulic machines, radial and axial -
Performance conversion method from model to prototype
(IEC 62097:2009)**

Machines hydrauliques,
radiales et axiales -
Méthode de conversion des performances
du modèle au prototype
(CEI 62097:2009)

Hydraulische Maschinen,
radial und axial -
Leistungsumrechnung
vom Modell zum Prototyp
(IEC 62097:2009)

This European Standard was approved by CENELEC on 2009-03-01. CENELEC members are bound to comply with the CEN/CENELEC Internal Regulations which stipulate the conditions for giving this European Standard the status of a national standard without any alteration.

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CENELEC

European Committee for Electrotechnical Standardization
Comité Européen de Normalisation Electrotechnique
Europäisches Komitee für Elektrotechnische Normung

Central Secretariat: avenue Marnix 17, B - 1000 Brussels

Foreword

The text of document 4/242A/FDIS, future edition 1 of IEC 62097, prepared by IEC TC 4, Hydraulic turbines, was submitted to the IEC-CENELEC parallel vote and was approved by CENELEC as EN 62097 on 2009-03-01.

The International Standard contains attached files in the form of Excel file. These files are intended to be used as complement and do not form an integral part of this publication.

The following dates were fixed:

- latest date by which the EN has to be implemented at national level by publication of an identical national standard or by endorsement (dop) 2009-12-01
- latest date by which the national standards conflicting with the EN have to be withdrawn (dow) 2012-03-01

Annex ZA has been added by CENELEC.

Endorsement notice

The text of the International Standard IEC 62097:2009 was approved by CENELEC as a European Standard without any modification.

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Annex ZA
(normative)

**Normative references to international publications
with their corresponding European publications**

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

NOTE When an international publication has been modified by common modifications, indicated by (mod), the relevant EN/HD applies.

<u>Publication</u>	<u>Year</u>	<u>Title</u>	<u>EN/HD</u>	<u>Year</u>
IEC 60193	1999	Hydraulic turbines, storage pumps and pump-turbines - Model acceptance tests	EN 60193	1999

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INTRODUCTION

0.1 General remarks

This International Standard establishes the prototype hydraulic machine efficiency from model test results, with consideration of scale effect including the effect of surface roughness.

Advances in the technology of hydraulic turbo-machines used for hydroelectric power plants indicate the necessity of revising the scale effect formula given in 3.8 of IEC 60193. [1]¹ The advance in knowledge of scale effects originates from work done by research institutes, manufacturers and relevant working groups within the organizations of IEC and IAHR. [1 - 7]

The method of calculating prototype efficiencies, as given in this standard, is supported by experimental work and theoretical research on flow analysis and has been simplified for practical reasons and agreed as a convention. [8 - 10] The method is representing the present state of knowledge of the scale-up of performance from model to a homologous prototype.

Homology is not limited to the geometric similarity of the machine components, it also calls for homologous velocity triangles at the inlet and outlet of the runner/impeller. [2] Therefore, compared to IEC 60193, a higher attention has to be paid to the geometry of guide vanes.

According to the present state of knowledge, it is certain that, in most cases, the formula for the efficiency step-up calculation given in the IEC 60193 and earlier standards, overstated the step-up increment of the efficiency for the prototype. Therefore, in the case where a user wants to restudy a project for which a calculation of efficiency step-up was done based on any previous method, the user shall re-calculate the efficiency step-up with the new method given in this standard, before restudying the project of concern.

This standard is intended to be used mainly for the assessment of the results of contractual model tests of hydraulic machines. If it is used for other purposes such as evaluation of refurbishment of machines having very rough surfaces, special care should be taken as described in Annex B.

Due to the lack of sufficient knowledge about the loss distribution in Deriaz turbines and storage pumps, this standard does not provide the scale effect formula for them.

An excel work sheet concerning the step-up procedures of hydraulic machine performance from model to prototype is indicated at the end of this Standard to facilitate the calculation of the step-up value.

0.2 Basic features

A fundamental difference compared to the IEC 60193 formula is the standardization of scalable losses. In a previous standard (see 3.8 of IEC 60193:1999 [1]), a loss distribution factor V has been defined and standardized, with the disadvantage that turbine designs which are not optimized benefit from their lower technological level.

This is certainly not correct, since a low efficiency design has high non-scalable losses, like incidence losses, whereby the amount of scalable losses is about constant for all manufacturers, for a given type and a given specific speed of a hydraulic machine.

This standard avoids all the inconsistencies connected with IEC 60193:1999. (see 3.8 of [1]) A new basic feature of this standard is the separate consideration of losses in specific hydraulic energy, disc friction losses and leakage losses. [5], [8 - 10]

¹ Numbers in square brackets refer to the bibliography.

Above all, in this standard, the scale-up of the hydraulic performance is not only driven by the dependence of friction losses on Reynolds number Re , but also the effect of surface roughness Ra has been implemented.

Since the roughness of the actual machine component differs from part to part, scale effect is evaluated for each individual part separately and then is finally summed up to obtain the overall step-up for a complete turbine. [10] For radial flow machines, the evaluation of scale effect is conducted on five separate parts; spiral case, stay vanes, guide vanes, runner and draft tube. For axial flow machines, the scalable losses in individual parts are not fully clarified yet and are dealt with in two parts; runner blades and all the other stationary parts inclusive.

The calculation procedures according to this standard are summarized in Clause 7 and Excel sheets are provided as an Attachment to this standard to facilitate the step-up calculation.

In case that the Excel sheets are used for evaluation of the results of a contractual model test, each concerned party shall execute the calculation individually for cross-check using common input data agreed on in advance.

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